



Integrating Oil Debris and Vibration Gear Damage Detection Technologies Using Fuzzy Logic

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Abstract

A diagnostic tool for detecting damage to spur gears was developed. Two different measurement technologies, wear debris analysis and vibration, were integrated into a health monitoring system for detecting surface fatigue pitting damage on gears. This integrated system showed improved detection and decision-making capabilities as compared to using individual measurement technologies. This diagnostic tool was developed and evaluated experimentally by collecting vibration and oil debris data from fatigue tests performed in the NASA Glenn Spur Gear Fatigue Test Rig. Experimental data were collected during experiments performed in this test rig with and without pitting. Results show combining the two measurement technologies improves the detection of pitting damage on spur gears.

Introduction

One technology area recommended for helicopter accident reduction is the design of helicopter Health Usage Monitoring Systems (HUMS) capable of predicting imminent equipment failure for on-condition maintenance and a more advanced system capable of warning pilots of impending equipment failure. Today's helicopter health monitoring systems (HUMS) are not at this level. Data collected by HUMS is often processed after the flight and plagued with high false alarm rates and undetected faults. The current fault detection rate of commercially available HUMS through vibration analysis is 70 percent [1]. False warning rates average 1 per hundred flight hours [2]. Often these systems are complex and require extensive interpretation by trained diagnosticians [3].

Transmission diagnostics are an important part of a helicopter HUMS because helicopters depend on the power train for propulsion, lift, and flight maneuvering. In order to predict transmission failures, the diagnostic tools used in the health monitoring system must provide real-time performance monitoring of aircraft operating parameters and must demonstrate a high level of reliability to minimize false alarms.

Various techniques exist for diagnosing damage in helicopter transmissions. The method most widely used involves vibration. Algorithms are developed, using vibration data collected from gearbox accelerometers, to detect when gear damage has occurred. Oil debris is also used to identify abnormal wear related conditions at an early stage. Oil debris monitoring for gearboxes consists mainly of off-line oil analysis, or plug type chip detectors. Although not commonly used for gear damage detection, many engines have on-line oil debris sensors for detecting the failure of rolling element bearings. These on-line, inductance type sensors count the number of particles, their approximate size, then calculate an accumulated mass.

Integrating the sensors into one system can potentially improve the detection capabilities and the probability that damage is detected. Recent investigations have shown the benefits of using an oil debris monitor with vibration data to improve current HUMS, but have not fully demonstrated a system with improved detection and decision-making capability when integrating the two measurement systems [4], [5].

The objective of the work reported herein is to improve the detection capability of vibration and oil based damage detection features by applying fuzzy logic analysis techniques to gear failure data collected from the NASA Glenn Spur Gear Fatigue Rig. A simple model was defined by the fuzzy rules and the membership functions for the experiments when pitting damage occurred. The ability to define valid ranges and limits for each membership function was found to be critical to the success of the model at predicting damage.

Vibration data were collected from accelerometers and used in previously validated gear vibration diagnostic algorithms. Oil debris data were collected using a commercially available in-line oil debris sensor. Oil debris and vibration data will be integrated using fuzzy logic analysis techniques. The goal of this research is to provide the end user with a simple tool to determine reliably the health of this geared system.

Experimental Investigation

Experimental data were recorded from 24 experiments performed in the Spur Gear Fatigue Test Rig at NASA Glenn Research Center. A sketch of the test rig is shown in Figure 1. The facility operates on the torque regenerative principle. Torque is applied by a hydraulic loading mechanism that twists one slave gear relative to its shaft. The power required to drive the system is only enough to overcome friction losses in the system [6]. The test gears are standard spur gears having 28 teeth, 3.50 inch (8.89 cm) pitch diameter, and 0.25 inch (0.635 cm) face width. The test gears are run offset to provide a narrow effective face width to maximize gear contact stress while maintaining an acceptable bending stress. Offset testing also allows four tests on one pair of gears. Two filters are located downstream of the oil debris monitor to capture the debris after it is measured by the sensor.

Fatigue tests were run in a manner that allows damage to be correlated to the oil debris sensor data. For these tests, run speed was 10,000 RPM and applied torque was 53 or 71 ft-lbs. (72 or 96 N•m). Prior to collecting test data, the gears were run for 1 hour at a torque of 10 ft-lbs. (14 N•m). Test gears were inspected periodically for fatigue damage throughout the duration of the test. When

damage was found, the damage was documented and correlated to the test data based on a reading number. Reading number refers to the once per minute data collection rate. Reading number is equivalent to minutes and can also be interpreted as mesh cycles equal to reading number times 10^4 . In order to document tooth damage, reference marks are made on the driver and driven gears during installation to identify tooth 1. The mating teeth numbers on the driver and driven gears are then numbered from this reference. Figure 2 identifies the driver and driven gear with the gearbox cover removed.

The principal focus of this research is the detection of pitting damage on spur gears. Pitting is a fatigue failure caused by exceeding the surface fatigue limit of the gear material. Pitting occurs when small pieces of material break off from the gear surface, producing pits on the contacting surfaces [7]. Gears are run until pitting occurs on one or more several teeth. Two levels of pitting were monitored, initial (pits less than 0.0397 cm diameter and cover less than 25 percent of tooth contact area) and destructive pitting (pits greater than 0.0397 cm diameter and cover greater than 25 percent of tooth contact area). If not detected in time, destructive pitting can lead to a catastrophic transmission failure if the gear teeth crack.

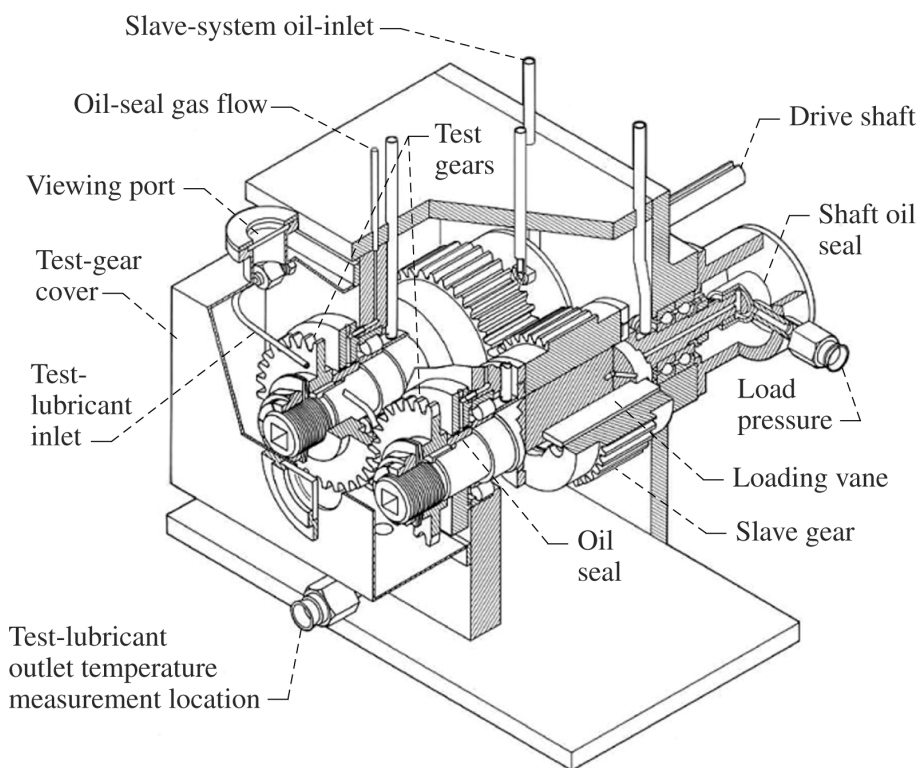


Figure 1: Spur Gear Fatigue Rig Gearbox

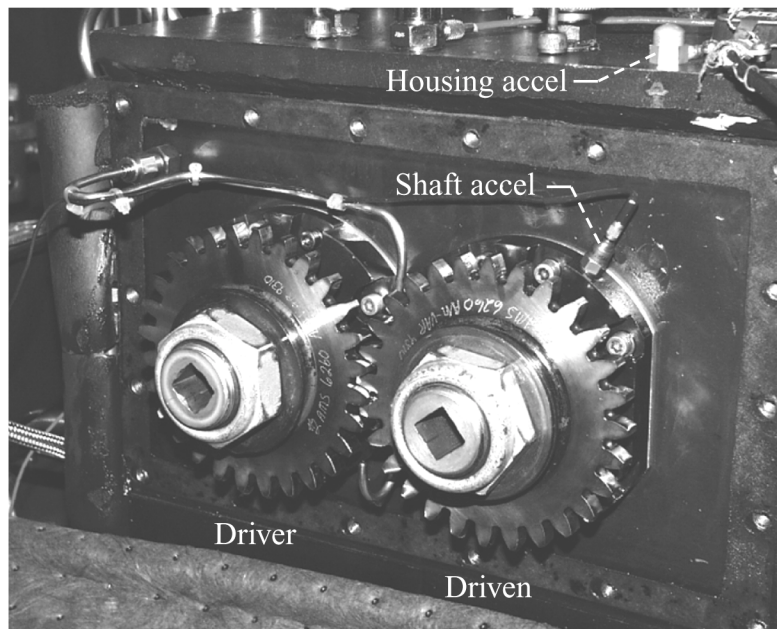


Figure 2: Spur Gear Fatigue Rig Gearbox With Cover Removed

Data were collected using vibration, oil debris, speed and pressure sensors installed on the test rig. Vibration was measured on the gear housing and at a support bearing location using miniature, lightweight, piezoelectric accelerometers. Location of both sensors is shown in Figure 2. These locations were chosen based on an analysis of optimum accelerometer locations for this test rig [8]. Oil debris data were collected using a commercially available oil debris sensor that measures the change in a magnetic field caused by passage of a metal particle where the amplitude of the sensor output signal is proportional to the particle mass. The sensor measures the number of particles, their approximate size (125 to 1000 microns) and calculates an accumulated mass [9]. Shaft speed was measured by an optical sensor once per revolution of the shaft. Load pressure was measured using a capacitance pressure transducer.

Oil debris sensor, speed, pressure, and raw vibration data were collected and processed in real-time using the data acquisition program ALBERT, Ames-Lewis Basic Experimentation in Real Time, co-developed by NASA Glenn and Ames. Oil debris and pressure data were recorded once per minute. Vibration and speed data were sampled at 200 KHz for one-second duration every minute. Vibration algorithms FM4 and NA4 Reset were calculated from this data and recorded every minute. Time-synchronous averaging was performed from the raw vibration data for 113 revolutions of the test gear. The signal time-synchronous average is obtained by taking the

average of the signal in the time domain with each record starting at the same point in the cycle as determined by the once per gear revolution tachometer signal [10]. The time-synchronous average data were then used to calculate the two vibration diagnostic parameters FM4 and NA4 Reset. FM4 and NA4 are dimensionless parameters with nominal values of approximately 3. When gear damage occurs, the value increases for both FM4 and NA4 [11].

FM4 was developed to detect changes in the vibration pattern resulting from fatigue damage on a limited number of teeth [12]. The theory behind FM4 is that for a gear in good condition, the difference signal would be noise with a Gaussian amplitude distribution. The standard deviation should be relatively constant, and normalized kurtosis indicates a value of three. When a tooth develops a major defect, a peak or series of peaks appear in the difference signal, causing the kurtosis value to increase [10]. One problem with the FM4 parameter is that it decreases in sensitivity as the number of peaks of similar magnitude increase beyond two. For this reason, NA4 was developed for failures that involve more than two teeth.

NA4 was developed to detect the onset of fatigue damage and to continue to react to the damage as it spreads [13]. However, it does not perform well under fluctuating load conditions. Preliminary tests found NA4 sensitive to minor changes in load. NA4 Reset was developed from NA4 for applications with load fluctuations [14].

The oil debris monitor records counts of particles in bins set at a particle size range. The particle size is measured in microns. For these experiments, 16 bins were defined. The range of the bin sizes in microns is shown in Table 1. Based on the bin configuration, the average particle size for each bin is used to calculate the cumulative mass for the experiment. Previous research verified accumulated mass is a good predictor of pitting damage and identified threshold limits that discriminate between stages of pitting on spur gears [15].

During each experiment, measurements from two accelerometers and an oil debris sensor were monitored and recorded for the occurrence of pitting damage. The data measured from the vibration and oil debris sensors during experiments with and without damage were used to identify membership functions to build a simple fuzzy logic model. Using this fuzzy logic model, and the vibration and oil debris data, threshold limits were defined that discriminate between different levels of pitting wear.

Discussion of Results

The analysis discussed in this section is based on data collected during 24 experiments, 15 of which had pitting damage occur. Video inspection images are available for 13 of the experiments with pitting damage, 2 were performed prior to installation of the video inspection system.

Table 2 is a summary of the experiments performed and a description of the damage. The second column lists the reading number the pitting was first observed via video or manual inspection. Video inspection was used during Experiments 1 to 6 and 18 to 24. Manual inspection was used for

experiments 7 to 17. The “oil debris” column is the amount of debris measured at this reading. The last reading collected for this experiment is listed in the fifth column. All gears were visually inspected at test completion and the damage description and amount of debris at this time are listed in the last 2 columns. The damage description gives the damage observed on the driver (Dr) and driven (Dn) gears. Damage is defined as initial pitting (ip), and destructive pitting (de) to the total number of teeth for each gear. For example, Dr: de 3t, ip allt, is driver gear had destructive pitting on 3 teeth and initial pitting on all of the teeth. A detailed description of the damage to each tooth was correlated with the video images for each experiment.

A representative sample of a detailed damage description for each tooth, and the images obtained from the video inspection system is shown in Table 3 and Figure 3. The damage progression video images of tooth 11 on the driver and driven gear for Experiment 2 are shown in this figure. The damage is only shown on less than half of the tooth because the test gears are run offset to provide a narrow effective face width to maximize gear contact stress.

Fuzzy logic techniques were applied to the oil debris and vibration data in order to build a simple model that discriminates between stages of pitting wear. Fuzzy logic applies fuzzy set theory to data, where fuzzy set theory is a theory of classes with unsharp boundaries and the data belongs in a set based on its degree of membership [16]. The degree of membership can be any value between 0 and 1. The advantage of applying fuzzy logic to condition based maintenance is that it is flexible, making allowances for unanticipated behavior.

Table 1: Oil Debris Particle Size Ranges

| Bin | Bin range, μm | Average | Bin | Bin range, μm | Average |
|-----|--------------------------|---------|-----|--------------------------|---------|
| 1 | 125–175 | 150 | 9 | 525–575 | 550 |
| 2 | 175–225 | 200 | 10 | 575–625 | 600 |
| 3 | 225–275 | 250 | 11 | 625–675 | 650 |
| 4 | 275–325 | 300 | 12 | 675–725 | 700 |
| 5 | 325–375 | 350 | 13 | 725–775 | 750 |
| 6 | 375–425 | 400 | 14 | 775–825 | 800 |
| 7 | 425–475 | 450 | 15 | 825–900 | 862.5 |
| 8 | 475–525 | 500 | 16 | 900–1016 | 958 |

Table 2: Summary of Experiments

| Experiment Number | Rdg Pitting First Observed | Damage Description | Oil Debris (mg) | Rdg at Test Completion | Damage Description | Oil Debris (mg) |
|-------------------|----------------------------|------------------------|-----------------|------------------------|---|-----------------|
| 1 | 14369 | Dr: de 1t Dn: de 1t | 15.475 | 15136 | Dr: de 3t, ip allt Dn: de3t | 36.108 |
| 2 | 2199 | Dr: Dn: de 1t | 8.934 | 2444 | Dr: de 2t, ip allt Dn de 3t, ip 3t | 26.268 |
| 3 | 2669 | Dr: de 2t Dn: de 2t | 8.690 | 3029 | Dr: de 3t, ip allt Dn: de 3t, ip3t | 14.148 |
| 4 | 2065 | Dr: de 3t Dn: | 12.132 | 4863 | Dr: de 7t, ip allt Dn de 3t, ip allt | 26.227 |
| 5 | 2566 | Dr: ip 2t Dn: | 7.413 | 4425 | Dr: de 11t, ip allt Dn de 10t, ip allt | 10.811 |
| 6 | 12061 | Dr: Dn: de 1t | 14.365 | 12368 | Dr: de 1t, ip allt Dn de 2t, ip allt | 22.851 |
| 7 | | | | 13716 | Dr: ip 1t Dn ip 1t | 3.381 |
| 8 | 5181 | Dr: ip 2t Dn: ip 3t | 6.012 | 5314 | Dr: de 6t, ip8t Dn de6t, ip7t | 19.101 |
| 9 | | | | 29866 | No damage | 2.359 |
| 10 | | | | 20452 | No damage | 5.453 |
| 11 | | | | 204 | No damage | 0.418 |
| 12 | | | | 15654 | No damage | 2.276 |
| 13 | | | | 25259 | No damage | 3.159 |
| 14 | | | | 5322 | No damage | 0 |
| 15 | | | | 21016 | No damage | 0.125 |
| 16 | | | | 380 | No damage | 0.099 |
| 17 | | | | 21066 | No damage | 0.064 |
| 18 | | | | 888 | Dr: de 6t, ip allt Dn de 4t, ip allt | 22.541 |
| 19 | | | | 199 | Dr: de 3t, ip allt Dn de 1t, ip allt | 11.230 |
| 20 | | | | 1593 | Dr: de 1t, ip allt Dn ip allt | 5.346 |
| 21 | 317 | Dr: de 1t Dn: de 1t | 4.04 | 514 | Dr: de 2t, ip allt Dn: de 2t, ip allt | 17.912 |
| 22 | | | | 838 | Dr: ip 5t Dn: de 3t, ip allt | 7.224 |
| 23 | | | | 10688 | Dr: de 2t, ip allt Dn: de 1t, ip allt | 6.399 |
| 24 | 7170 | Dr: de 1t Dn: | 6.186 | 7224 | Dr: de 1t, ip allt Dn: ip allt | 9.681 |

Note: ip=initial pitting; de=destructive pitting; Dr=driver gear; Dn=driven gear; Xt=number of teeth with damage

Table 3: Damage Description for Experiment 2

| Reading Number Run Time (min) | Damage Description | Teeth Damaged on Driver Gear | Teeth Damaged on Driven Gear |
|----------------------------------|--|---------------------------------|---------------------------------|
| 1573 | Run-in Wear | All | All |
| 2199 | Wear Destructive Pitting | All | All 11 |
| 2296 | Wear Destructive Pitting | All | All 10, 11 |
| 2444 | Wear Initial Pitting Destructive Pitting | All All 10, 11 | All 10, 11, 14 10, 11, 14 |

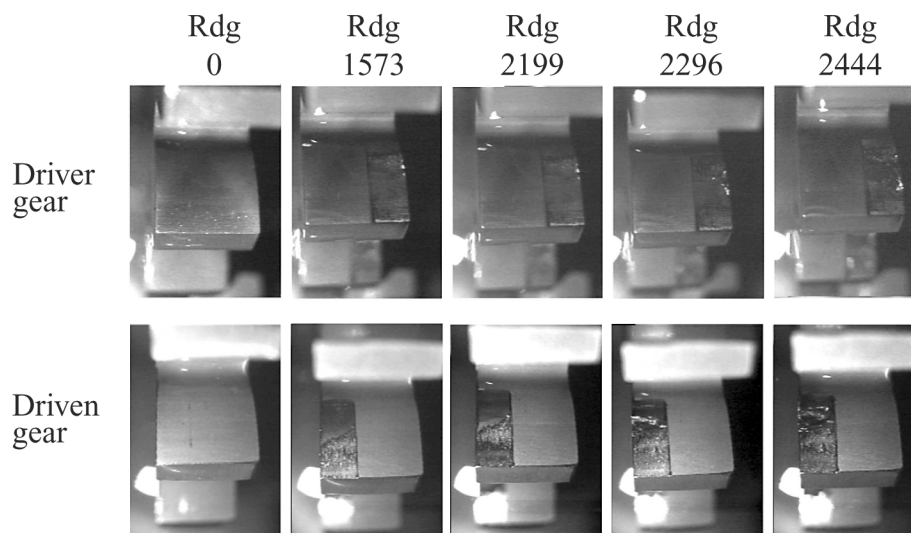


Figure 3: Damage Progression of Driver/Driven Tooth 11 for Experiment 2

Mamdani's fuzzy inference system is the most common seen fuzzy methodology and used for this application [17]. It is based on the paper on fuzzy algorithms for decision processes [18]. In the Mamdani type inference systems the output membership functions are fuzzy sets. The process is detailed below [19]:

1. fuzzify inputs or fuzzification: converts each piece of input data to degrees of membership by a lookup in one of several membership functions.
2. apply fuzzy operator: AND = minimum; OR = maximum
3. apply implication methods: apply weight to rule; output fuzzy set is truncated and scaled.
4. aggregate all outputs: aggregation is the process by which fuzzy sets represent the outputs of each rule and are combined into a single fuzzy set.
5. defuzzify: output is a single number. Middle of maximum (the average of the maximum value of the output set)

Defining the fuzzy logic model requires inputs (damage detection features), outputs (state of gear), and rules. Commercially available software was used to build the model because it was a convenient tool for mapping an input space to an output space and creating and editing fuzzy inference systems [20]. Input space for this model was defined as damage low (DL), damage medium (DM), and damage high (DH), indicated by the following features: oil debris mass (DL, DM, DH), NA4 Reset (DL, DH), and FM4 (DL, DH). The output space for this model was defined as the state of the gear. The 3 states of the gear to predict with this model were identified as: O.K. (no gear damage); Inspect (initial/destructive pitting); Shutdown due to damage (severe destructive pitting). The Mean of the maximum (MOM) was chosen as the defuzzification method. MOM was chosen because it gave the most plausible results for this application. The MOM method finds the output with the maximum membership and takes the x-axis average of all points with this maximum

membership value. If there is more than one point that has maximum degree of membership, the mean of the points are taken. The membership functions were based on the data collected during experiments 1 to 17.

For the oil debris sensor, membership values were based on the accumulated mass and the amount of damage observed the teeth via video or visual inspection. Membership values are defined for 3 levels of damage: damage low (DL), damage medium (DM), and damage high (DH) and are shown in Figure 4. The process used to define membership functions for the oil debris sensor are published in an earlier paper and indicate accumulated mass is a good predictor of pitting damage on spur gears and fuzzy logic is a good technique for setting threshold limits that discriminates between states of pitting wear [15].

FM4 is the vibration feature developed to detect changes in the vibration pattern due to damage on a limited number of teeth. When gear damage occurs, the FM4 value increases, and then decreases as it progresses to a number of teeth. FM4 was calculated for the accelerometers located on the bearing support and the housing. The maximum value of FM4 measured by the two accelerometers was used for further analysis. FM4 membership values were defined by looking at the maximum FM4 value within each inspection interval. Membership values are defined for 2 levels of damage: damage low (DL) and damage high (DH). The membership function for FM4 is shown in Figure 5. Due to FM4's insensitivity to

damage progression, logic is programmed into the model to freeze the FM4 when it exceeds 7.68.

NA4 Reset, like FM4, is less sensitive to damage as it progresses to a number of teeth and increases in severity. Although, the magnitude of NA4 Reset is significantly larger than FM4 when pitting damage begins to occur, like FM4, NA4 reset decreases as damage progresses to a number of teeth. NA4 was calculated for the accelerometers located on the bearing support and the housing. The maximum value of NA4 measured by the two accelerometers was used for further analysis. NA4 membership values were defined by looking at the maximum NA4 value within each inspection interval. Membership values are defined for 2 levels of damage: damage low (DL) and damage high (DH). The membership function for NA4 is shown in Figure 6. Due to NA4's insensitivity to damage progression, logic is programmed into the model to freeze the NA4 when it exceeds 12.60.

The degree of membership for the output of the fuzzy model is shown in Figure 7. The output is the status or state of the gear: O.K. (no gear damage); Inspect (initial pitting); Damage (destructive pitting). The rules defined for the model are listed in Table 4. Using the membership values and rules for the vibration and oil debris features, and the Mean of the Maximum (MOM) fuzzy logic defuzzification method, a simple fuzzy logic model was developed. The input/output data to the fuzzy model for each experiment will be discussed in the following paragraphs.

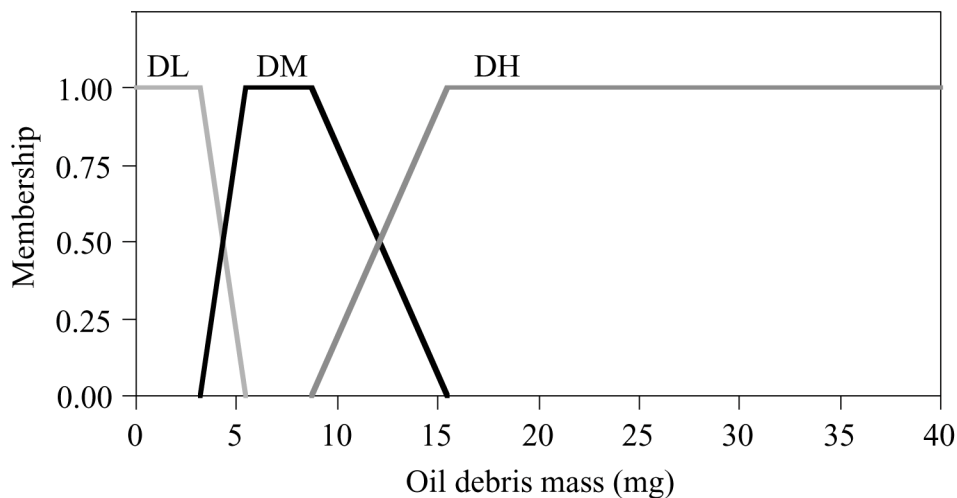


Figure 4: Membership Values for Oil Debris Feature

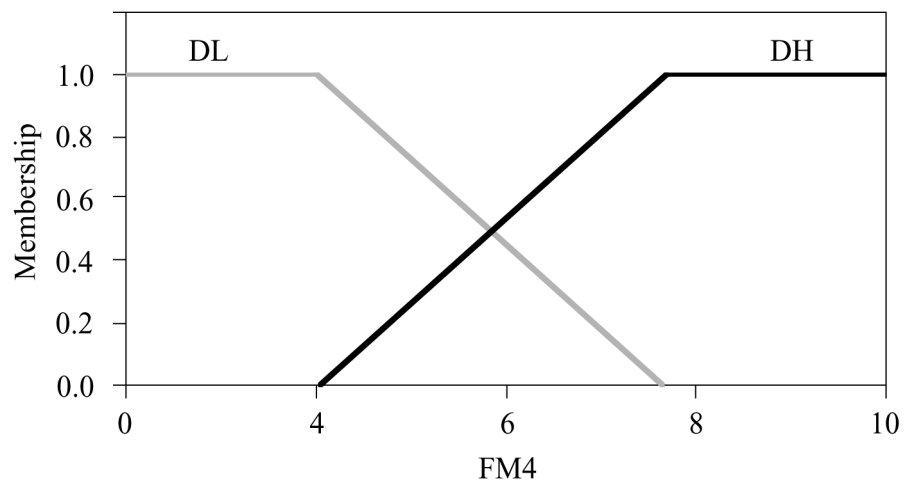


Figure 5: Membership Values for FM4 Feature

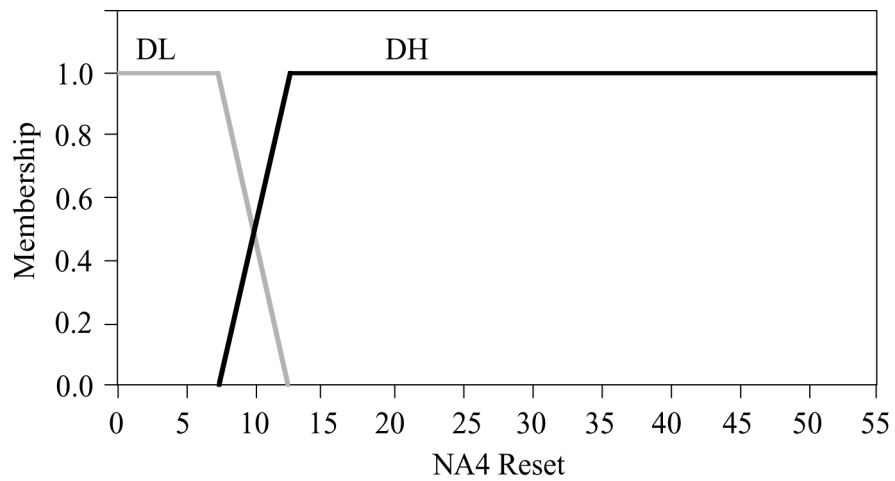


Figure 6: Membership Values for NA4 Feature

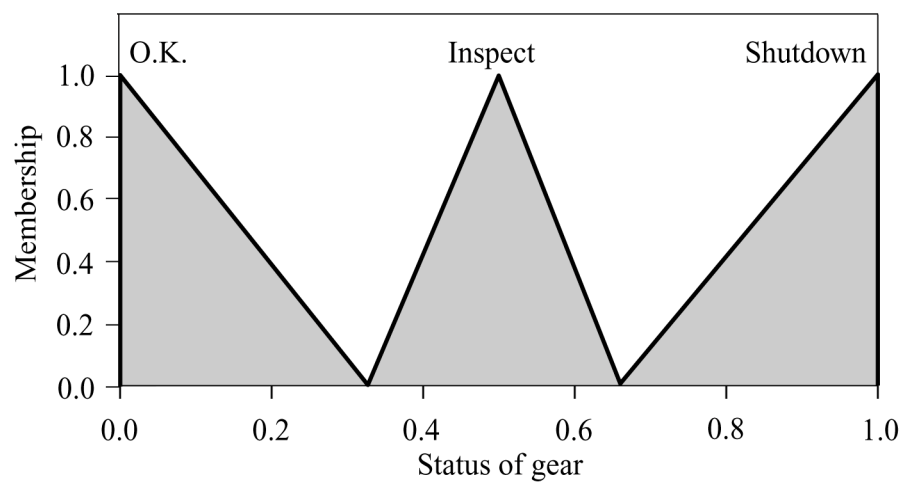


Figure 7: Output of Fuzzy Logic Model

Table 4: Rules for Fuzzy Logic Model

| | |
|-----|---|
| 1. | If (FM4 is DL) and (NA4 is DL) and (debris is DL) then (output is O.K.) |
| 2. | If (FM4 is DH) and (NA4 is DH) and (debris is DH) then (output is SHUTDOWN) |
| 3. | If (FM4 is DL) and (NA4 is DL) and (debris is DM) then (output is INSPECT) |
| 4. | If (FM4 is DL) and (NA4 is DH) and (debris is DL) then (output is O.K.) |
| 5. | If (FM4 is DL) and (NA4 is DL) and (debris is DH) then (output is INSPECT) |
| 6. | If (FM4 is DH) and (NA4 is DL) and (debris is DL) then (output is O.K.) |
| 7. | If (FM4 is DH) and (NA4 is DL) and (debris is DM) then (output is INSPECT) |
| 8. | If (FM4 is DH) and (NA4 is DH) and (debris is DL) then (output is INSPECT) |
| 9. | If (FM4 is DH) and (NA4 is DL) and (debris is DH) then (output is SHUTDOWN) |
| 10. | If (FM4 is DH) and (NA4 is DH) and (debris is DM) then (output is INSPECT) |
| 11. | If (FM4 is DL) and (NA4 is DH) and (debris is DH) then (output is SHUTDOWN) |
| 12. | If (FM4 is DL) and (NA4 is DH) and (debris is DM) then (output is INSPECT) |

Figures 8 through 12 are representative plots for 5 of the 20 experiments. Each figure is comprised of 2 plots. The plot on the top is a plot of the 3 features measured during each experiment. FM4 and NA4 Reset correspond to the left Y-axis, the accumulated mass measured by the oil debris sensor corresponds to the right Y-axis. These features are input into a simple fuzzy logic model. The plot on the bottom is the fuzzy logic output. The triangles on the X-axis correspond to when video inspection was performed. The background colors in different shades of gray indicate the O.K., inspect, and shutdown due to damage states.

A short description of Figures 8 through 12 will follow. The results of experiment 2 are plotted on Figure 8. Destructive pitting was first observed on one tooth of the driven gear at reading 2199 and the output plot indicates to inspect the gears. As the damage increases, the inspect changes to shutdown for this experiment. Figure 9 presents the results of experiment 3. Destructive pitting was first observed on two teeth of both the driven and driver gear at reading 2669 and the output plot indicates to inspect the gears. As the damage increases, the inspect changes to shutdown for this experiment. Experiment 12 is plotted on Figure 10. No damage occurred during this experiment, and the output plot remains in the green region. Experiment 8 is plotted on Figure 11. Initial pitting was first observed on 2 driver teeth and 3 driven teeth at reading 5181 and the output plot indicates to inspect the gears. As the damage increases, the inspect changes to shutdown for this experiment. Experiment 18, not used to define the membership functions, is plotted on Figure 12. At test completion, destructive pits were observed on 6 of the driver teeth and 4 of the driven teeth.

After review of the data from these experiments, the advantage of integrating the features of different measurement technologies into a simple

fuzzy logic model is evident. The output gives clear information to the end user when making a decision based on the data. The model developed incorporates the expert knowledge of the diagnostician into a system that can be used to make clear decisions on the status of the geared system.

Several observations are worth noting after careful analysis of the data. The first is that the oil debris feature was more reliable than the vibration features for detecting pitting fatigue failure of spur gears. The vibration features were more sensitive to the environment (operational effects, location, sampling rates, etc.) and these sensitivities were more difficult to quantify or correct for in the field.

Another observation is that a technique for setting accurate threshold limits for vibration algorithms was not clearly defined in the literature [10], [11], [13], [21], [22], [23]. It appears to be a trial and error method that changes for each experiment and each test rig. This makes it very difficult to quantify the false alarms and missed hits using the individual algorithms. If the threshold limits for the vibration algorithms are set at any number above the nominal value of 3.0, the false alarms would dominate [10], [12], [13], [21].

In comparison, the thresholds for this analysis were determined based on membership functions defined for 17 experiments with varied operational conditions. The process used to define membership functions for the vibration algorithms was an attempt to intelligently define threshold limits. Setting thresholds using membership functions gives the end user more flexibility in defining threshold limits based on levels of damage. However, this method also has its limitations in that it requires several sets of damage data to refine the limits.

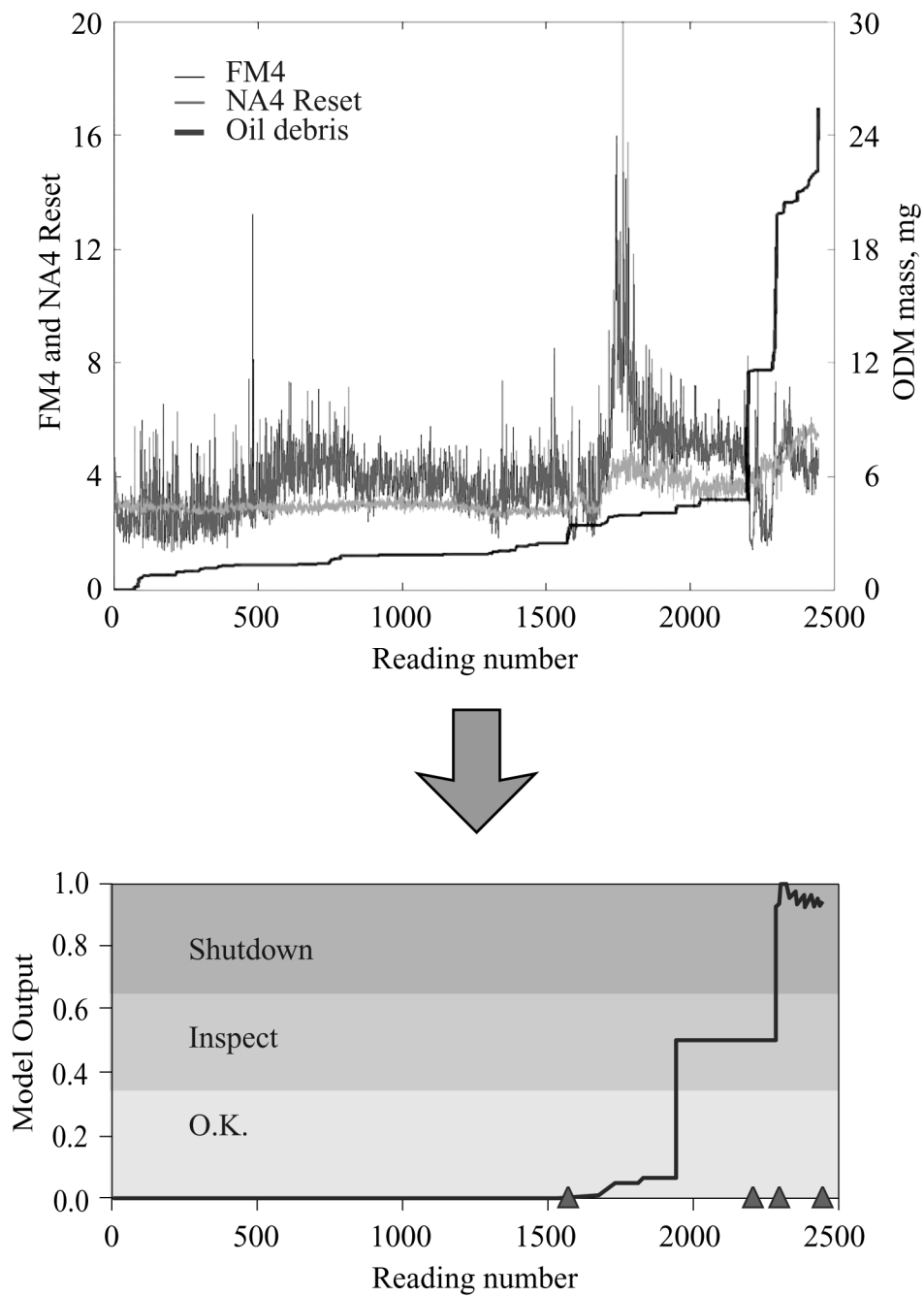


Figure 8: Experiment 2 Features and Model Output

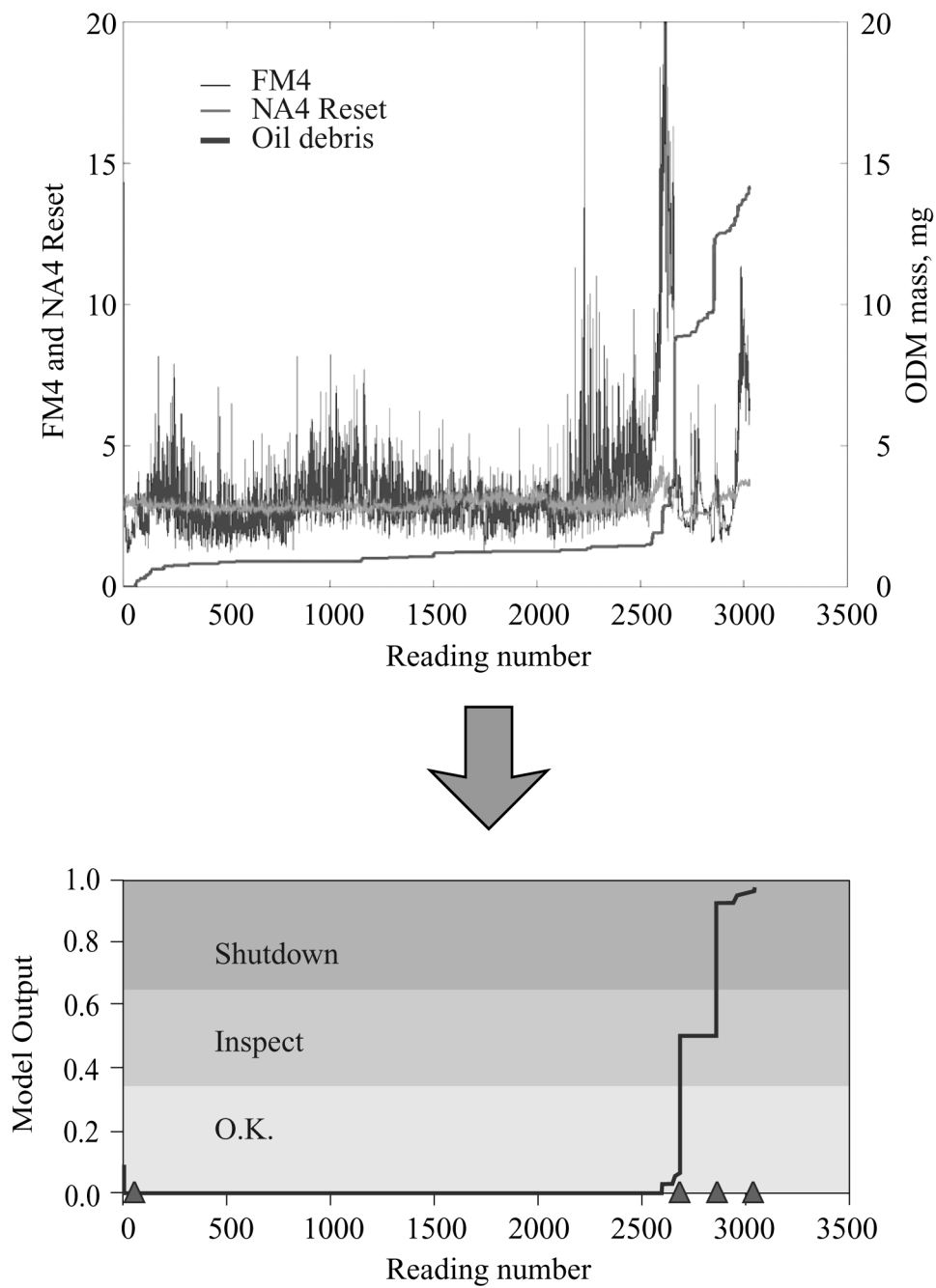


Figure 9: Experiment 3 Features and Model Output

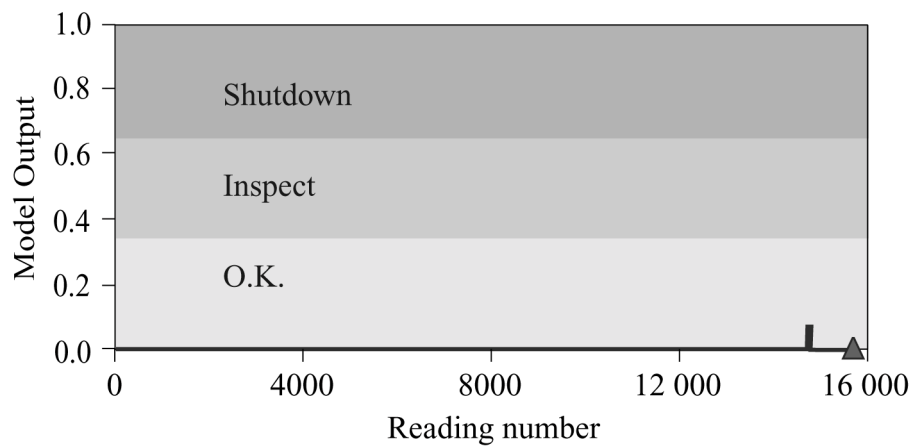
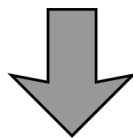
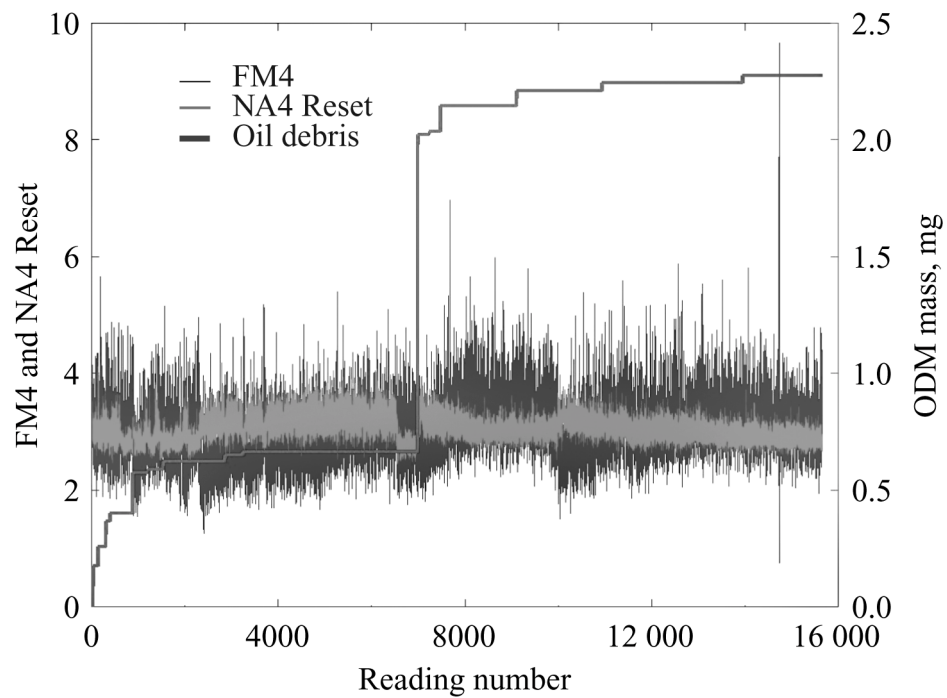


Figure 10: Experiment 12 Features and Model Output

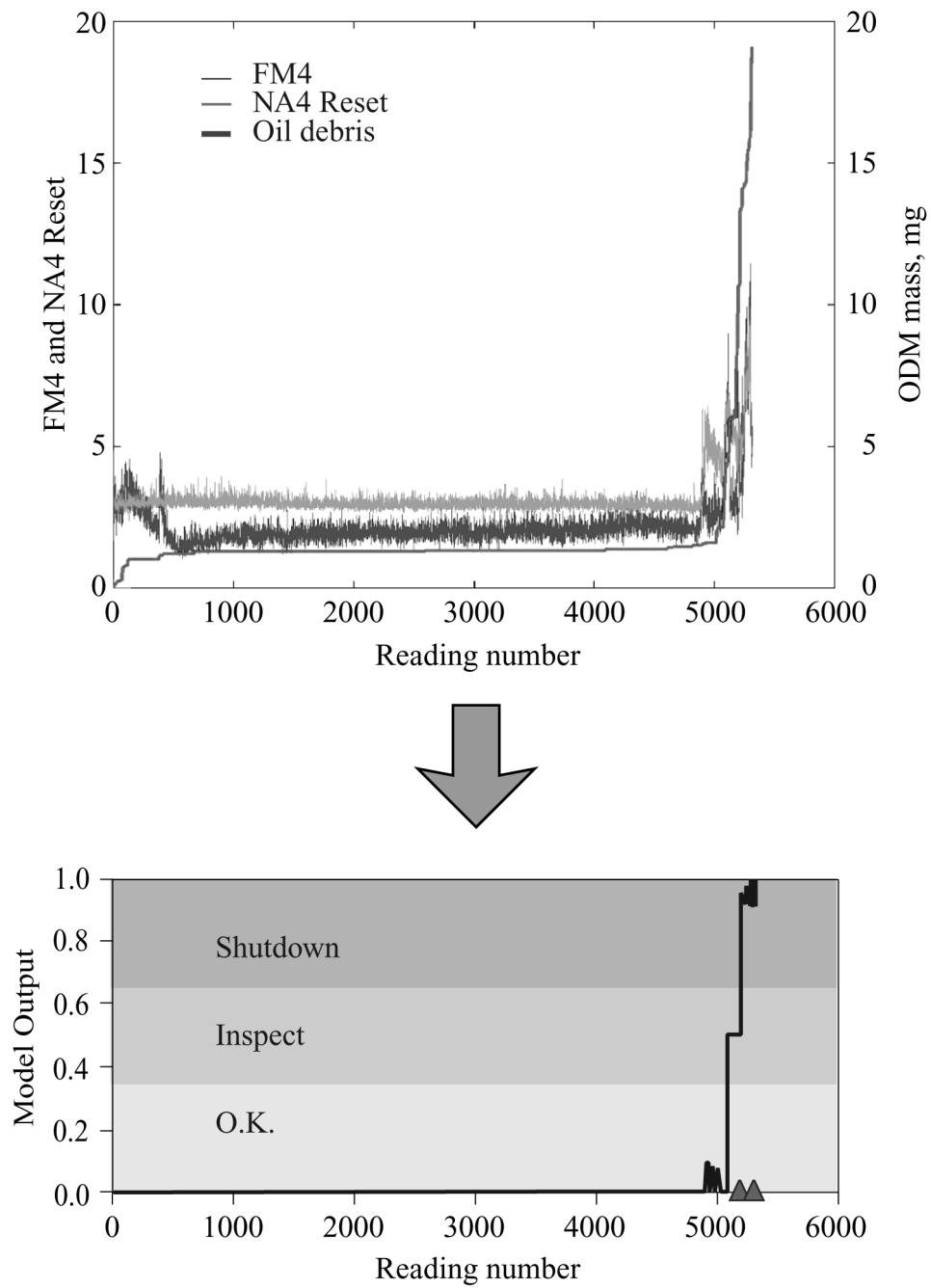


Figure 11: Experiment 8 Features and Model Output

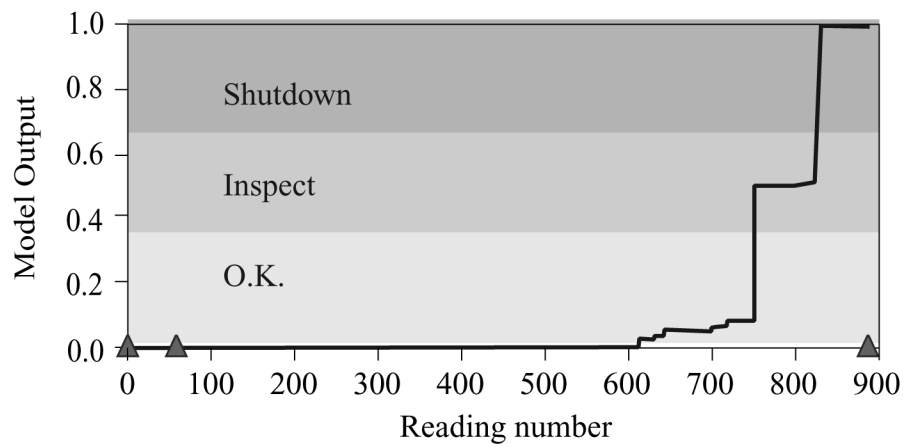
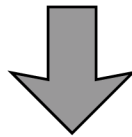
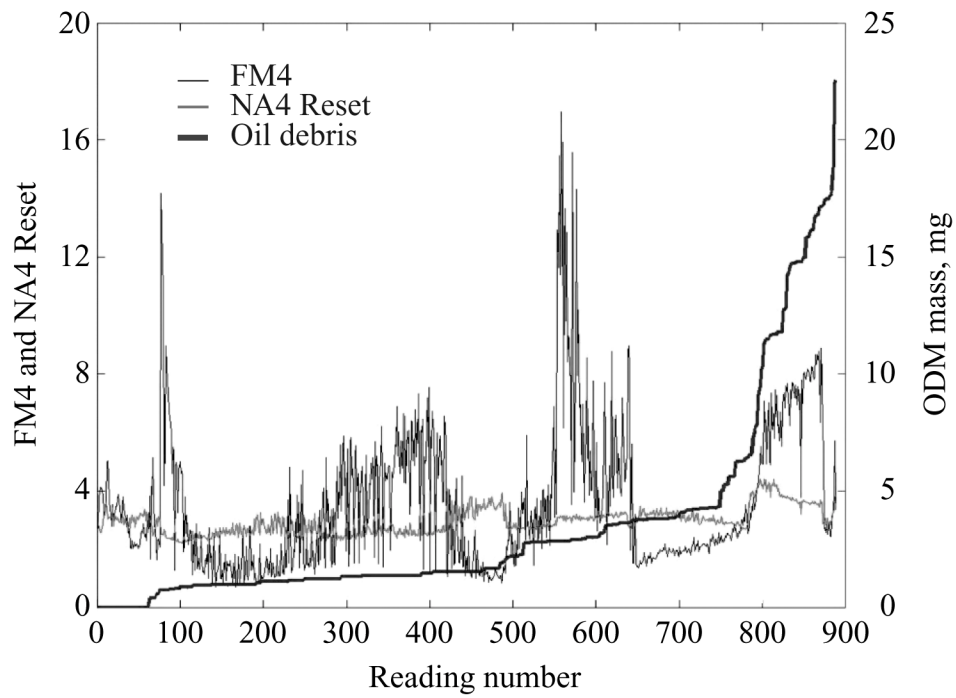


Figure 12: Experiment 18 Features and Model Output

Conclusions

The integration of two measurement technologies, oil debris analysis and vibration, results in a system with improved damage detection and decision-making capabilities. Vibration and oil debris data were collected from experiments in the NASA Glenn Spur Gear Fatigue Rig. Using fuzzy logic techniques applied to the oil debris and vibration data, a simple system model was developed that discriminates between stages of pitting wear. Results indicate combining the vibration and oil debris measurement technologies improves the detection of pitting damage on spur gears. As a result of this research, the diagnostic tools used for damage detection in the NASA Glenn Spur Gear Fatigue Rigs have been significantly improved.

Several other findings were made that will impact the development of health monitoring tools for geared systems. The first being, oil debris analysis is more reliable than vibration analysis for detecting pitting fatigue failure of spur gears. The second finding is that some vibration algorithms are as sensitive to operational effects as they are to damage. The third finding is that vibration algorithms FM4 and NA4 Reset do not indicate damage progression, but the increase in oil debris mass is related to damage progression. The fourth finding is that clear threshold limits must be established by the developer of the diagnostic tool if it is to be applied to other systems. The development of membership functions for each parameter will improve this process. It also enables the end user to replace these parameters with their own by adjusting the membership functions. The fifth finding relates to the human factors aspect of diagnostic tool development. As a diagnostician, it is important to identify the end user of the diagnostic tool early on in the process so that he or she can use the diagnostic tool to make clear decisions on the health of the geared system.

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